

Solved Problems Part One on

Adaptive Control

4th Year Petroleum Systems

and Control Engineering

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Transfer Function to State Space Representations

Consider the dynamic of a system given in the following Transfer Function:

$$G(s) = \frac{y(s)}{u(s)} = \frac{6}{s^3 + 6s^2 + 11s + 6}$$

Then $y''' + 6y'' + 11y' + 6y = 6u$

Note: Number of states = the degree of the differential equation

We have 3 states:

$$x_1 = y, \quad x_2 = y', \quad x_3 = y''$$

taking the derivative with respect to time

$$x_1' = y' = x_2, \quad x_2' = y'' = x_3$$

$$\dot{x}_3 = y''' = -6y - 11\dot{y} - 6\ddot{y} + 6u$$

$$\dot{x}_3 = y''' = -6x_1 - 11x_2 - 6x_3 + 6u$$

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix} = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -6 & -11 & -6 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ 6 \end{bmatrix} u$$

$$y = x_1$$

$$y = \begin{bmatrix} 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix} + 0u$$

$$\dot{x} = A x + B u \quad (\text{state equation})$$

$$y = C x + D u \quad (\text{output equation})$$

where

$$A = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ -6 & -11 & -6 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 0 \\ 6 \end{bmatrix}$$

$$C = [1 \quad 0 \quad 0], \quad D = 0$$

$$\dot{x} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{bmatrix}, \quad x = \begin{bmatrix} x_1 \\ x_2 \\ x_3 \end{bmatrix}$$

Note: for (n) states with (m) inputs and (r) output:

$A = n \times n$ in this case 3×3 (Dimension)

$B = n \times m$ in this case 3×1 (Dimension)

$C = r \times n$ in this case 1×3 (Dimension)

$X = n \times 1$ in this case 3×1 (Dimension)

$D = r \times m$ in this case 1×1 (Dimension)

State Space to Transfer Function Representations

Consider the dynamic of a system with State Space representation:

$$\dot{x} = A x + B u$$

$$y = C x + D u$$

Where y is the system output and u is the system input

the dynamic of a system can be presented by Transfer Function as:

$$G(s) = \frac{y(s)}{u(s)}$$

Taking Laplace Transform for the state space equations:

$$s X(s) - x(0) = A X(s) + B U(s) \quad (1)$$

$$Y(s) = C X(s) + D U(s) \quad (2)$$

From equation (1) assuming relaxed system ($x(0) = 0$)

$$s X(s) - A X(s) = B U(s)$$

$$(sI - A)X(s) = B U(s)$$

$$X(s) = (sI - A)^{-1} B U(s) \quad (3)$$

Substitute (3) in (2):

$$Y(s) = C (sI - A)^{-1} B U(s) + D U(s)$$

$$G(s) = \frac{Y(s)}{U(s)} = C (sI - A)^{-1} B + D$$

And can be written as:

$$G(s) = \frac{Q(s)}{(sI - A)}$$

(sI - A) is the characteristic polynomial of G(s). In other words
The eigenvalues of (A) = The poles of (G)

Question (1): Write the Transfer Function of a system given its state space representation as:

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ 1 \end{bmatrix} u$$

$$y = \begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

Solution:

$$A = \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix}, \quad B = \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$C = \begin{bmatrix} 1 & 0 \end{bmatrix}, \quad D = 0$$

$$G(s) = C (sI - A)^{-1} B + D$$

$$sI - A = \begin{bmatrix} s & 0 \\ 0 & s \end{bmatrix} - \begin{bmatrix} 0 & 1 \\ -2 & -3 \end{bmatrix} = \begin{bmatrix} s & -1 \\ 2 & s + 3 \end{bmatrix}$$

$$(sI - A)^{-1} = \frac{\begin{bmatrix} s + 3 & 1 \\ -2 & s \end{bmatrix}}{s^2 + 3s + 2}$$

$$G(s) = \begin{bmatrix} 1 & 0 \end{bmatrix} \frac{\begin{bmatrix} s + 3 & 1 \\ -2 & s \end{bmatrix}}{s^2 + 3s + 2} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$G(s) = \frac{1}{s^2 + 3s + 2} \begin{bmatrix} 1 & 0 \end{bmatrix} \begin{bmatrix} s + 3 & 1 \\ -2 & s \end{bmatrix} \begin{bmatrix} 0 \\ 1 \end{bmatrix}$$

$$G(s) = \frac{1}{s^2 + 3s + 2}$$

Question (2):

Calculate the desired closed loop equation in discrete form for a second order system with $\omega_n = 10$, $\zeta = 0.7$ and the sampling time is 0.1 second.

Solution:

The standard form of a second order Transfer Function is:

$$\frac{U(s)}{R(s)} = \frac{\omega_n^2}{s^2 + 2 \zeta \omega_n s + \omega_n^2} = \frac{100}{s^2 + 2 * 0.7 * 10 s + 100}$$

$$\frac{U(s)}{R(s)} = \frac{100}{s^2 + 14 s + 100} = \frac{100}{(s + P1)(s + P2)}$$

$$P1, P2 = \frac{-14 \pm \sqrt{14^2 - 4 * 100}}{2} = -7 \pm j7.14$$

$$\frac{U(s)}{R(s)} = \frac{100}{(s + 7 + j7.14)(s + 7 - j7.14)}$$

$$Z = e^{T s} = e^{T P_{1,2}} = e^{0.1 * (-7 \pm j7.14)} = e^{-0.7 \pm j0.714}$$

$$Z_{1,2} = 0.496 * (0.755 \pm j0.655) = 0.375 \pm j0.325$$

$$T = (z - z_1)(z - z_2)$$

$$= (z - 0.375 + j0.325)(z - 0.375 - j0.325)$$

$$T = z^2 - 0.75 z + 0.246$$

$$T = 1 - 0.75 z^{-1} + 0.246 z^{-2}$$

Question (3):

Derive the PID controller equation in z-domain using backward approximation.

Solution: The PID equation in S-plane is:

$$G_c(s) = K_p + \frac{K_i}{s} + K_d s$$

Using backward difference, the PID equation for $s = \frac{z-1}{Ts z}$ is:

$$G_c(z) = K_p + \frac{K_i T_s z}{z-1} + \frac{K_d (z-1)}{T_s z}$$

Assume $T_s = 1$ (special case)

$$G_c(z) = \frac{K_p (z-1) z + K_i z^2 + K_d (z-1)^2}{z (z-1)}$$

$$G_c(z) = \frac{K_p z^2 - K_p z + K_i z^2 + K_d z^2 - 2 z K_d + K_d}{z^2 - z}$$

$$G_c(z) = \frac{(K_p + K_d z + K_i)z^2 - (K_p + 2 K_d)z + K_d}{z^2 - z} * \frac{z^{-2}}{z^{-2}}$$

$$G_c(z) = \frac{(K_p + K_d z + K_i) - (K_p + 2 K_d)z^{-1} + K_d z^{-2}}{1 - z^{-2}}$$

$$G_c(z) = \frac{g_0 + g_1 z^{-1} + g_2 z^{-2}}{1 - z^{-2}}$$

Where

$$g_0 = (Kp + Ki + Kd) \quad (1)$$

$$g_1 = -(Kp + 2 Kd) \quad (2)$$

$$g_2 = Kd \quad (3)$$

Substitute (3) into (2), we get:

$$Kp = -g_1 - 2 Kd$$

$$Kp = -g_1 - 2 g_2 \quad (4)$$

Substitute (3) and (4) in (1), we get:

$$g_0 = -g_1 - 2 g_2 + g_2 + Ki$$

$$Ki = g_0 + g_1 + g_2$$

Pole assignment control:

It is a classical approach to controller specifications. The design of a feedback controller has two main aims:

- 1- the first is to modify in some way the dynamic response of a system.
- 2- The second is to reduce the sensitivity of the system output to Disturbances.

Suppose we have a system with a continuous time transfer function:

$$Y(s) = \frac{f}{1 + \alpha.s} u(s) \quad (1)$$

where α is the time constant of the system and f is the system gain.

The discrete-time domain transfer function is:

$$Y(z) = \frac{Z^{-1} \cdot b}{1 - a \cdot Z^{-1}} U(z) \quad (2)$$

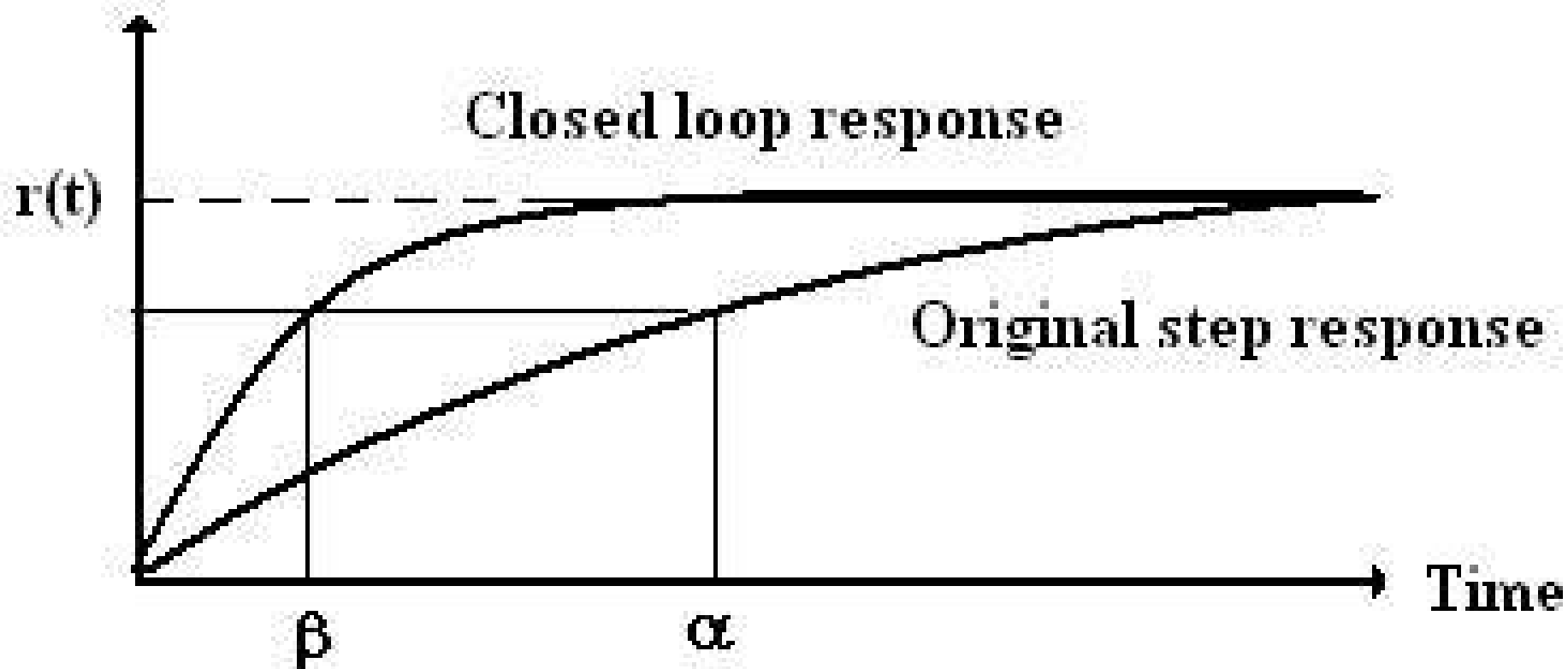
$$\text{where } a = e^{-T/\alpha} \text{ and } b = (1 - a) \cdot f \quad (3)$$

and T is the sampling time.

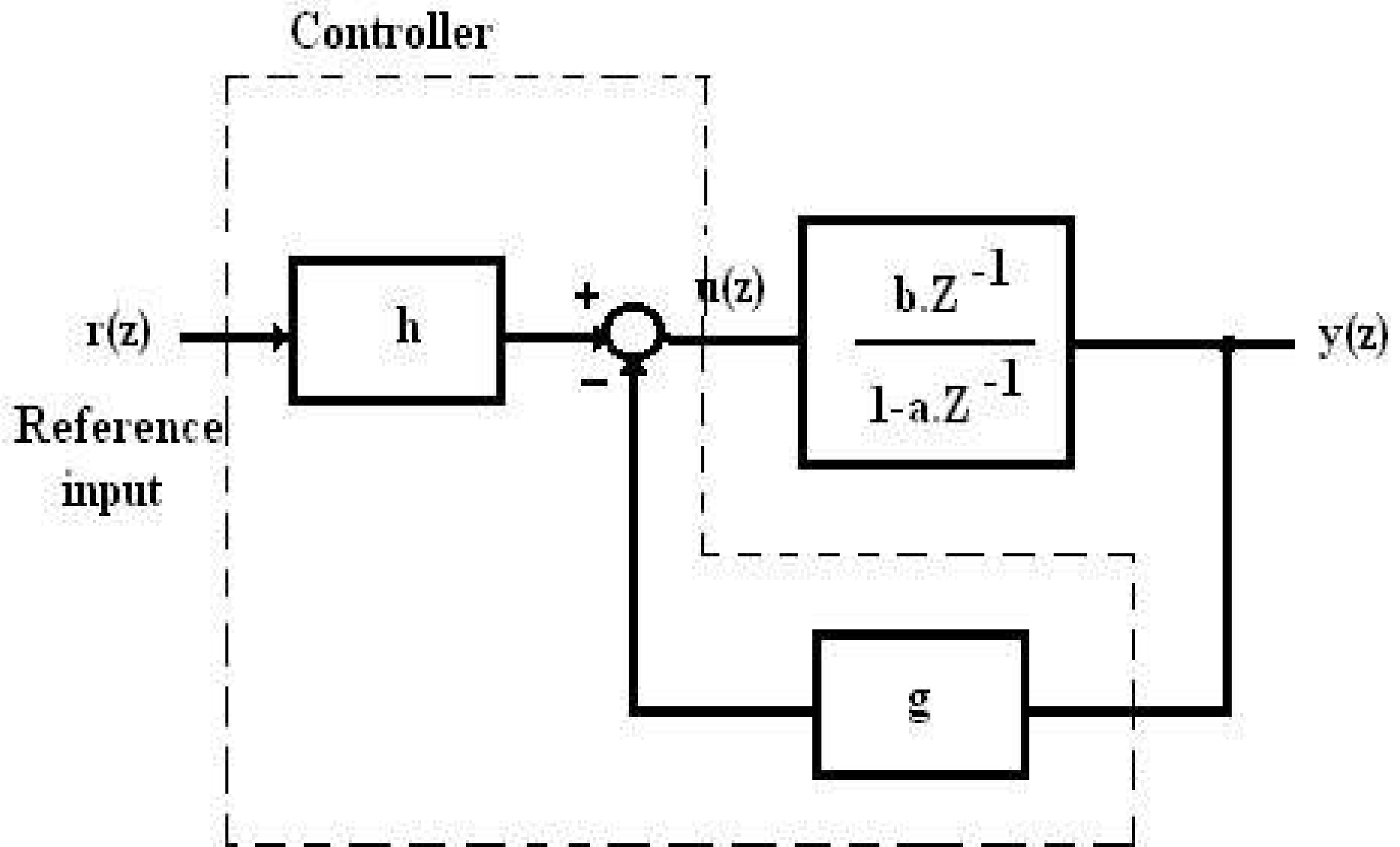
Common control objective would be to use a feedback controller to alter the speed of the system response. I.e. the original system might have a certain speed of response to a sudden change in $u(z)$ associated with time constant α , but it is required to change this response rate to one associated with time constant β .

In terms of the system model, equation 2 changing the speed of response corresponds to later the value of a in the denominator in some way assigning it another value corresponding to faster speed of response.

Also, it is required to ensure that the output $y(t)$ come into correspondence with reference signal $r(t)$.



To do this, propose the feedback controller structure with the following block diagram:



Where: $\mathbf{u}(Z) = -\mathbf{g}.y(Z) + \mathbf{h}.r(Z)$ (4)

By combining Eq. (2) and Eq. (4), this gives the closed loop transfer function:

$$y(z) = \frac{\mathbf{b.h.Z}^{-1}}{1 - (\mathbf{a} - \mathbf{b.g}).Z^{-1}} r(Z) \quad (5)$$

Notice that the closed loop speed of structure is determined by $(a-b.g)$ instead of a . In other words, the open loop system pole $Z=a$ has been assigned (placed) to a closed loop position $Z=a-b.g$. Hence, we can specify the location of the poles as a design parameter. Suppose we required that closed loop pole at $Z=t_1$, then:

$$\mathbf{t}_1 = \mathbf{a} - \mathbf{b.g} \quad (6)$$

$$\mathbf{g} = \frac{\mathbf{a} - \mathbf{t}_1}{\mathbf{b}} \quad \text{(Design equation)} \quad (7)$$

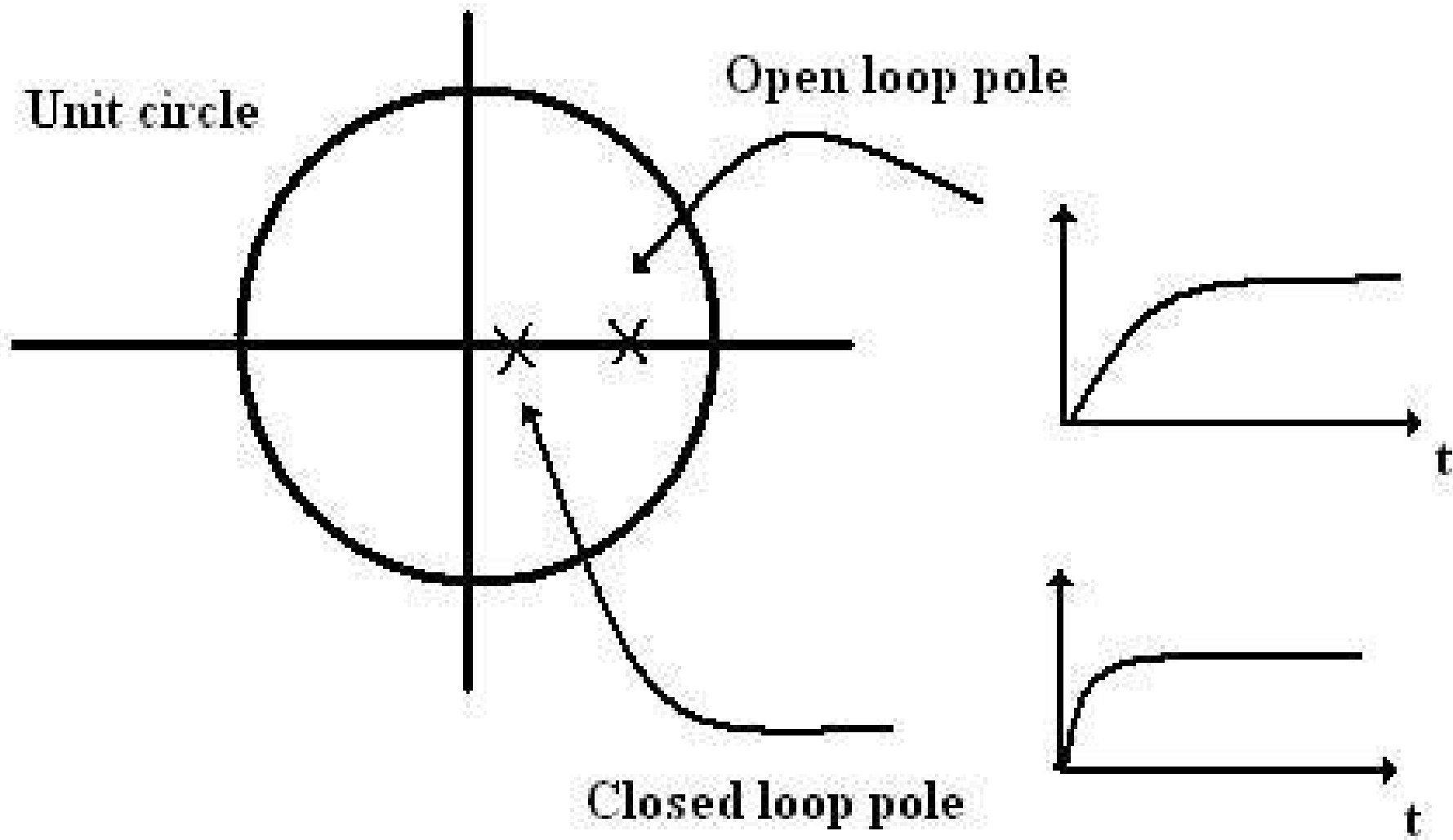
The remaining controller objective is to ensure that steady state (when $r(z)$ is constant), the output $y(z)=r(z)$.

This is done by selecting the controller parameter h such that the closed loop transfer function is equal to one at zero frequency. (Zero frequency means $s \rightarrow 0$ or $Z \rightarrow 1$). Using equation 5:

$$\frac{\mathbf{b.h}}{\mathbf{1 - (a - b.g)}} = \mathbf{1} \quad (8)$$

$$\mathbf{or\ h = \frac{1 - (a - b.g)}{b}} \quad (9)$$

Equation (9) is the design equation for selecting h so that $y(z)$ tracks $r(z)$ at steady state conditions. The key idea of pole assignment is to shift the open loop poles to some desired set of closed loop poles.



Question (4):

Design a digital controller for the open loop discrete transfer function of

$G_{OL}(z) = \frac{2}{z-1}$, so that the digital closed loop transfer function has a zero steady state error with characteristic equation of $T = 1 - 0.5z^{-1}$.

Solution:

The open loop transfer function can be re-written as: $G_{OL}(z) = \frac{2z^{-1}}{1-z^{-1}}$

The closed loop transfer function is shown below:

$$y(z) = \frac{2.h.z^{-1}}{1 - (1 - 2.g).z^{-1}} r(z)$$

By equating c\cs. equation with T, this gives:

$$1-2.g=0.5$$

$$2g=0.5$$

$$g=0.25$$

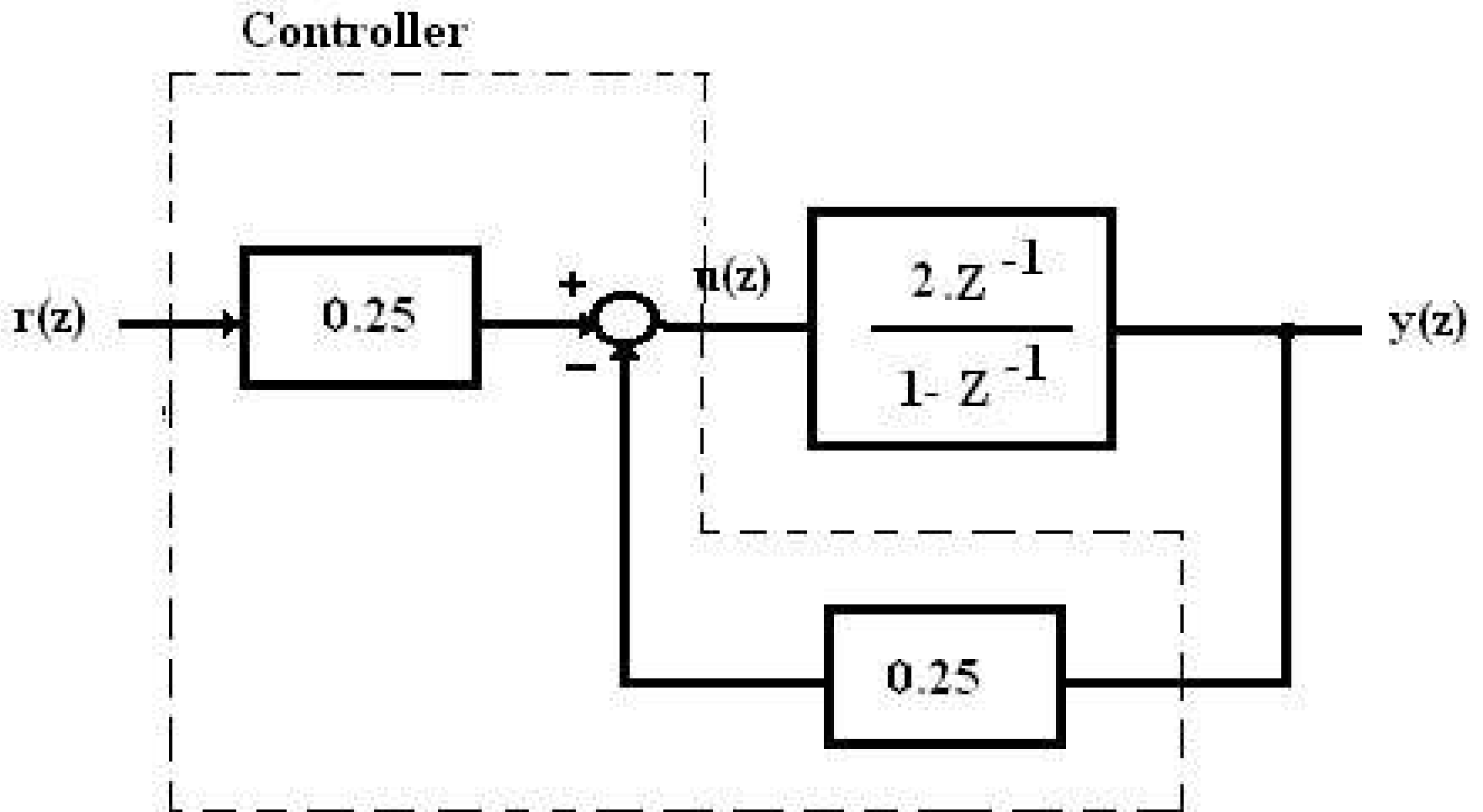
For zero steady state error,

$$\lim_{z \rightarrow 1} \frac{y(z)}{r(z)} = 1 = \frac{2.h}{1 - 0.5}$$

$$2.h=1-0.5$$

$$2.h=0.5$$

$$h=0.25$$



$$y(z) = \frac{0.5 \cdot Z^{-1}}{1 - 0.5 \cdot Z^{-1}} r(z)$$

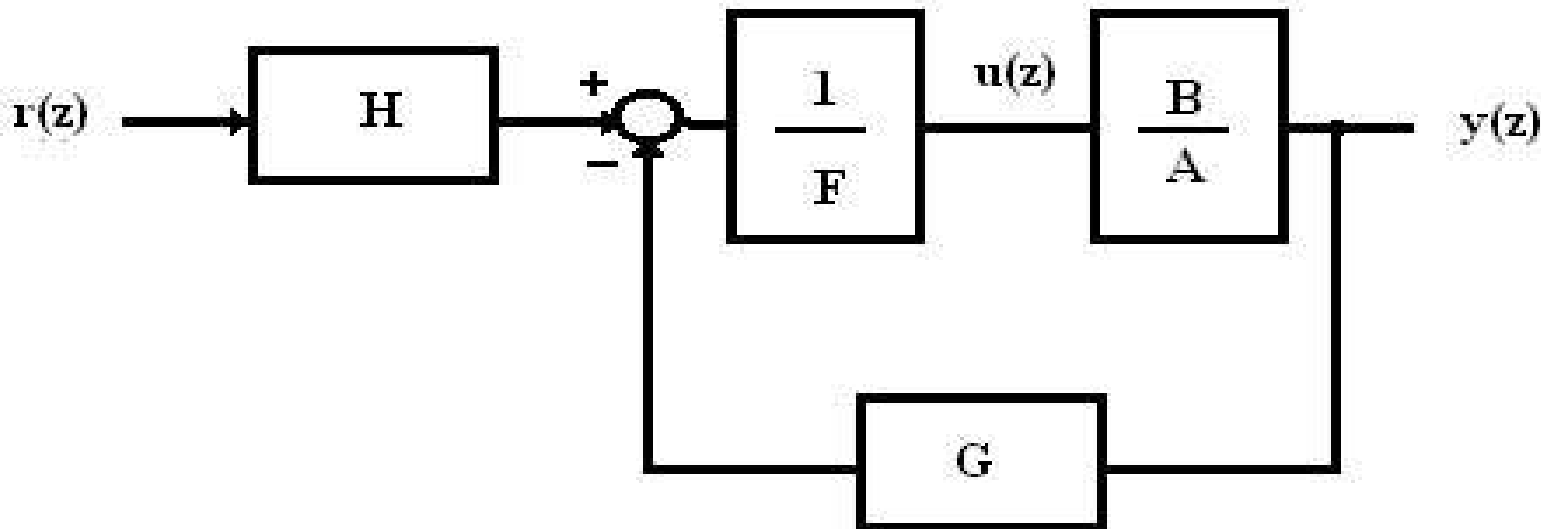
Question (5):

For the following model:
$$y(z) = \frac{Z^{-1} + 2Z^{-2}}{1 - 1.7Z^{-1} + 0.72Z^{-2}} u(z)$$

Design a digital controller using pole placement that satisfies the following Closed loop characteristic equation: $T = 1 - 0.904Z^{-1}$.

Solution:

The block diagram of the closed loop system is:



The closed loop Transfer Function is:

$$y(z) = \frac{B H}{F A + B G} r(z)$$

The design equation is:

$$F A + B G = T \quad (1)$$

$n_t = 1$, $n_b = 2$, $n_a = 2$, $n_f = n_b - 1 = 2 - 1 = 1$, $n_g = n_a - 1 = 2 - 1 = 1$

$n_t \leq n_a + n_b - 1$; $n_t \leq 2 + 2 - 1$; $n_t \leq 3$

The condition of n_t is satisfied

$$F = 1 + f_1 z^{-1}$$

$$G = g_0 + g_1 z^{-1}$$

Substitute in equation (1):

$$\begin{aligned}
& (1 + f_1 z^{-1})(1 - 1.7 z^{-1} + 0.72 z^{-2}) \\
& + (z^{-1} + 2 z^{-2})(g_0 + g_1 z^{-1}) = 1 - 0.904 z^{-1} \\
& 1 - 1.7 z^{-1} + 0.72 z^{-2} + f_1 z^{-1} - 1.7 f_1 z^{-2} + 0.72 f_1 z^{-3} \\
& + g_0 z^{-1} + g_1 z^{-2} + 2 g_0 z^{-2} + 2 g_1 z^{-3} = 1 - 0.904 z^{-1} \\
& 1 + (-1.7 + f_1 + g_0)z^{-1} + (0.72 - 1.7 f_1 + g_1 + 2 g_0)z^{-2} \\
& + (0.72 f_1 + 2 g_1)z^{-3} = 1 - 0.904 z^{-1}
\end{aligned}$$

$$z^{-1} : -1.7 + f_1 + g_0 = -0.904 \quad (2)$$

$$z^{-2} : 0.72 - 1.7 f_1 + g_1 + 2 g_0 = 0 \quad (3)$$

$$z^{-3} : 0.72 f_1 + 2 g_1 = 0 \quad (4)$$

From equation (2)

$$f_1 + g_0 = 0.796 \quad (5)$$

From equation (3)

$$-1.7 f_1 + 2 g_0 + g_1 = -0.72 \quad (6)$$

From equation (4)

$$0.72 f_1 + 2 g_1 = 0 \quad (7)$$

Write equations (5-7) in matrix forms

$$\begin{bmatrix} 1 & 1 & 0 \\ -1.7 & 2 & 1 \\ 0.72 & 0 & 2 \end{bmatrix} \begin{bmatrix} f_1 \\ g_0 \\ g_1 \end{bmatrix} = \begin{bmatrix} 0.796 \\ -0.72 \\ 0 \end{bmatrix} \equiv \begin{bmatrix} 1 & b_1 & 0 \\ a_1 & b_2 & b_1 \\ a_2 & 0 & b_2 \end{bmatrix} \begin{bmatrix} f_1 \\ g_0 \\ g_1 \end{bmatrix} = \begin{bmatrix} t_1 - a_1 \\ t_2 - a_2 \\ 0 \end{bmatrix}$$

A θ_c b

$$A \theta_c = b \quad \text{then} \quad \theta_c = A^{-1} b \quad (8)$$

Calculate the A^{-1} using matrix theory

$$A^{-1} = \begin{bmatrix} 0.4926 & -0.2463 & 0.1232 \\ 0.5074 & 0.2463 & -0.1232 \\ -0.1773 & 0.0887 & 0.4557 \end{bmatrix}$$

$$\theta_c = \begin{bmatrix} f_1 \\ g_0 \\ g_1 \end{bmatrix} = \begin{bmatrix} 0.5695 \\ 0.2265 \\ -0.205 \end{bmatrix}$$

$$F = 1 + 0.5695 z^{-1}$$

$$G = 0.2265 - 0.205 z^{-1}$$

The closed loop Transfer Function is:

$$\frac{y(z)}{r(z)} = \frac{(z^{-1} + 2 z^{-2}) H}{1 - 0.904 z^{-1}}$$

Given that steady state error is zero; using final value theorem

$$\lim_{z \rightarrow 1} \left(\frac{(z^{-1} + 2z^{-2}) H}{1 - 0.904z^{-1}} \right) = 1 \quad \text{then} \quad \frac{(1+2)h}{1-0.904}$$

$$h = \frac{1 - 0.904}{3} = 0.032 \quad H \text{ becomes } h \text{ for constant value}$$

Another solution: Assume the requirement to cancel the zero (z+2)

$$\text{let } H = \frac{z}{z+2} = \frac{1}{1+2z^{-1}}$$

$$\frac{y(z)}{r(z)} = \frac{(z^{-1} + 2z^{-2})}{(1 - 0.904z^{-1})(1 + 2z^{-1})} = \frac{z^{-1}(1 + 2z^{-1})}{(1 - 0.904z^{-1})(1 + 2z^{-1})}$$

$$\frac{y(z)}{r(z)} = \frac{z^{-1}}{1 - 0.904z^{-1}}$$